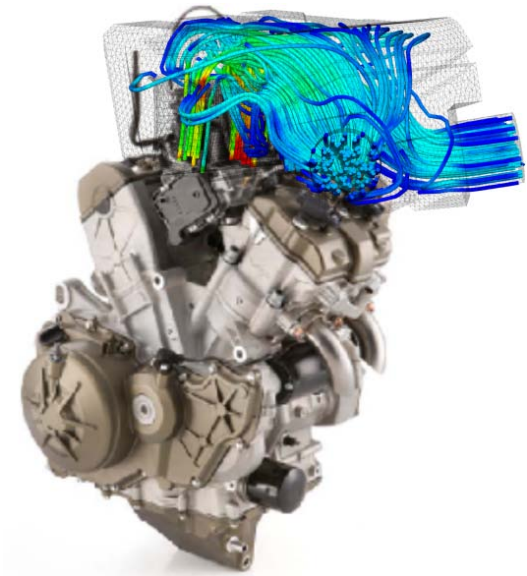


# 1D-3D THERMO – FLUID DYNAMIC SIMULATION OF A HIGH PERFORMANCE MOTORBIKE ENGINE



**L. Zucchi**

Piaggio & C. S.p.A. - Sede operativa Noale

T. Cerri, G. Montenegro

Department of Energy, Politecnico di Milano



POLITECNICO DI MILANO  
Internal Combustion Engine  
Group



PIAGGIO & C.s.p.a.

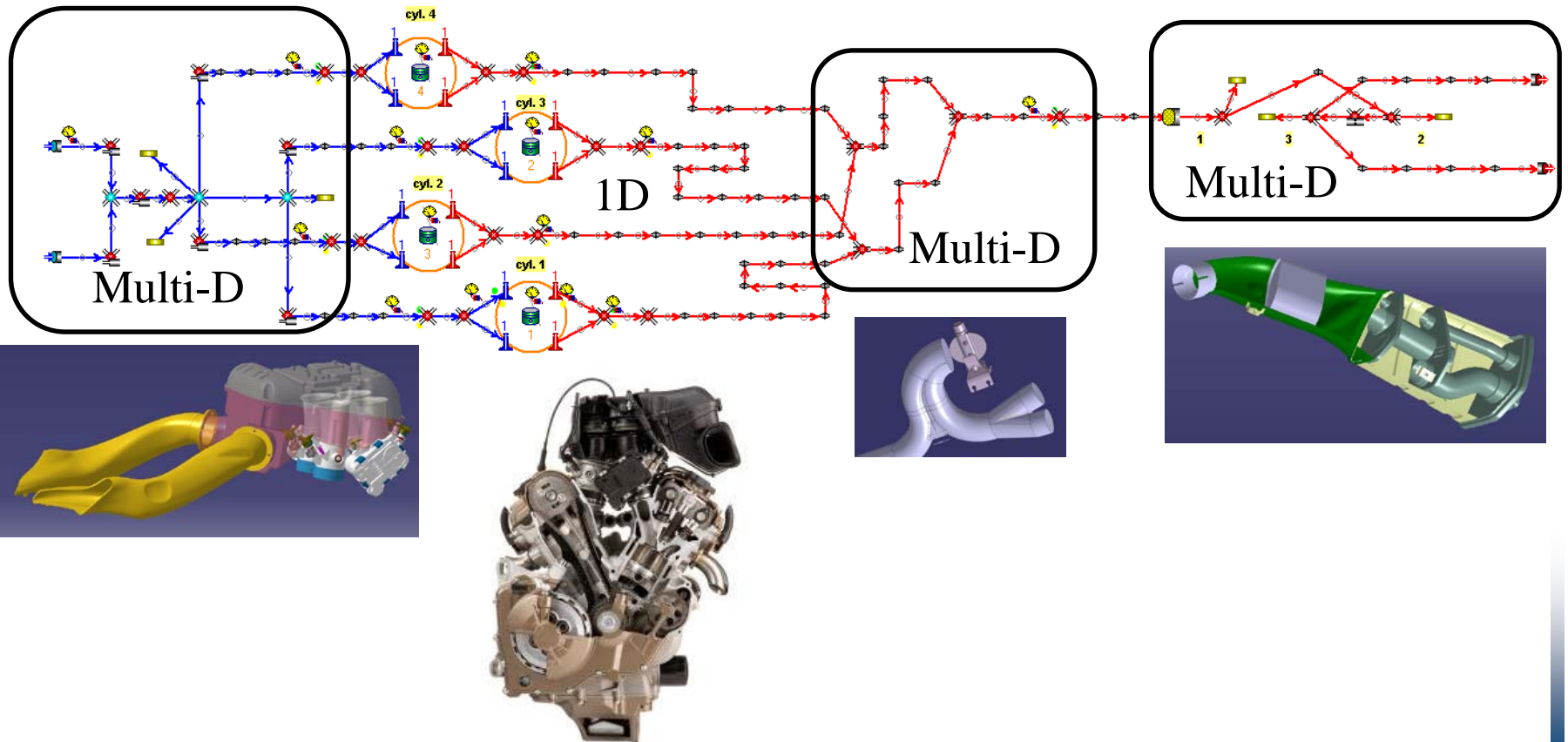


# Summary

- Introduction
- Engine characteristics
- Experimental apparatus
- Intake system analysis
- Exhaust system analysis
- Prediction of volumetric efficiency
- Conclusions
- Work in progress

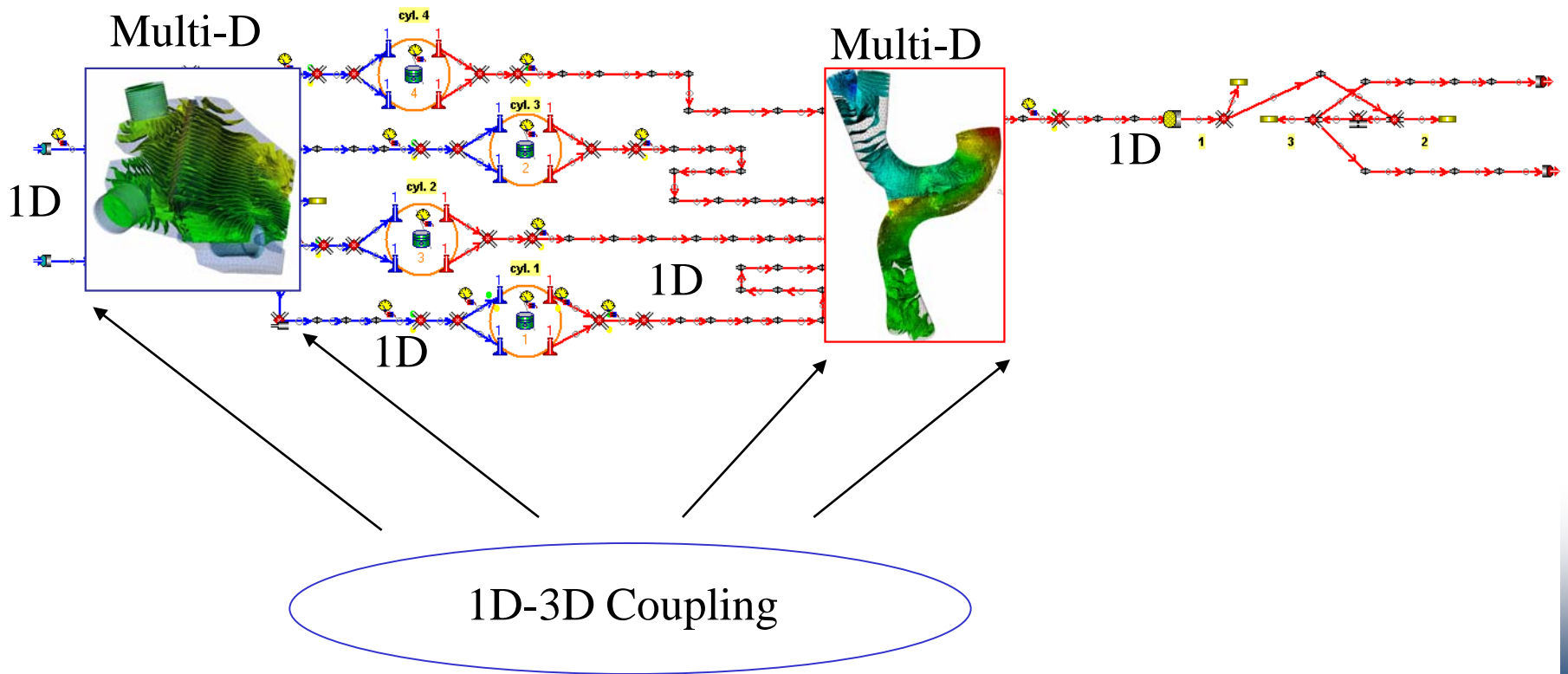
# Introduction

- High performance engine: accurate optimization of all components
  - Reduction of fluid dynamics losses and optimization of dynamic effects
  - Improve volumetric efficiency



# Introduction

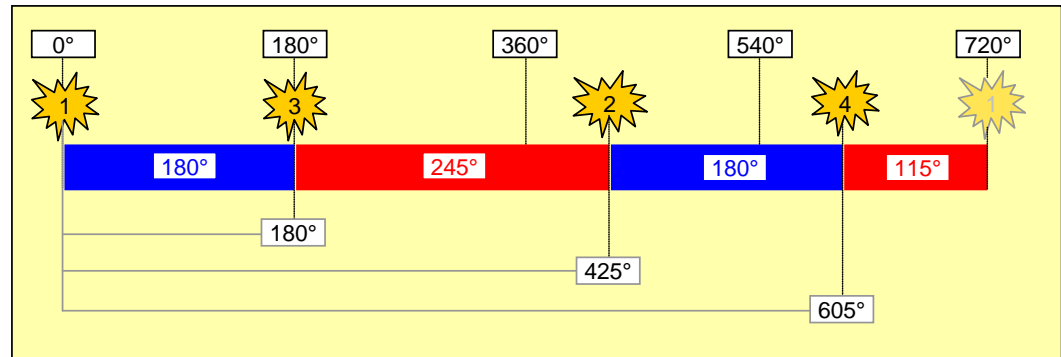
- 1D–3D simulation of intake and exhaust systems with multiple 3D domains (airbox and exhaust junction).



# Engine characteristics



## Firing Order



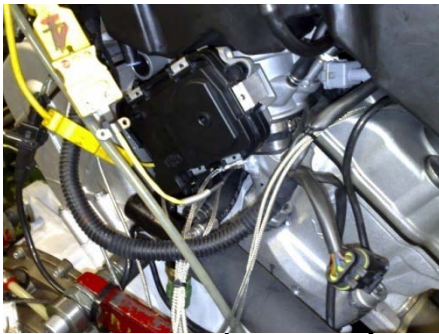
Engine type	Spark Ignition
Cylinder layout	V4 - 65°
Total displacement [dm <sup>3</sup> ]	0.9996
Bore [m]	0.078
Stroke [m]	0.0523
Compression ratio	13:1
Number of valves per cylinder	4
Air metering	Naturally aspirated
Injection system	PFI

# Experimental Apparatus

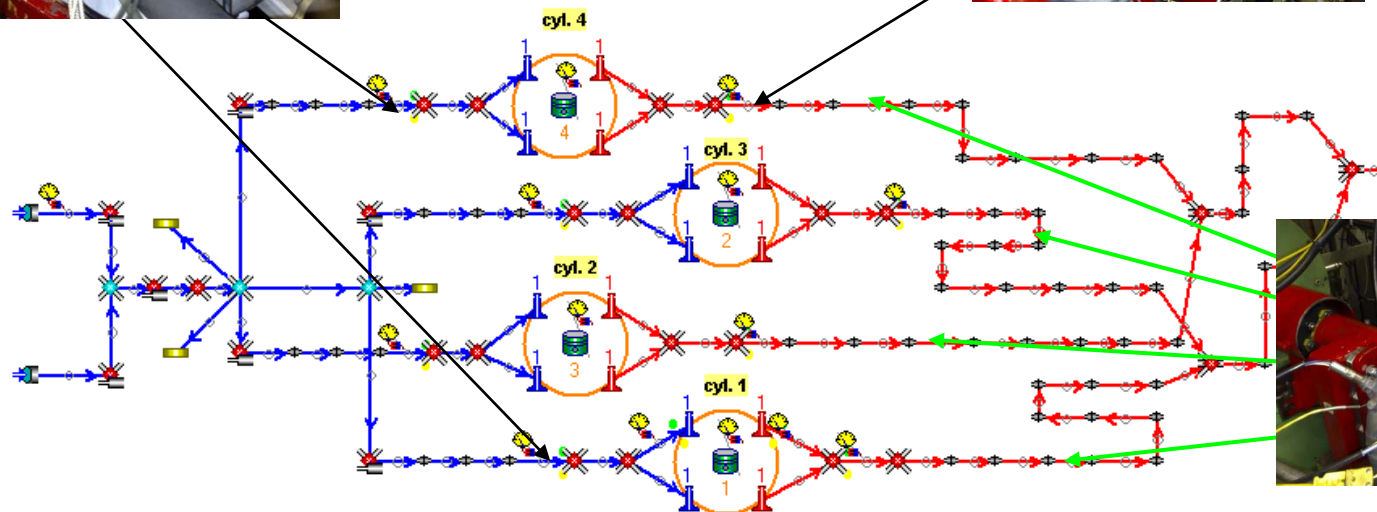
- **Istantaneous pressure**
  - Piezo – resistive transducer Kulite ETL 173-190 (intake)
  - Piezo – resistive transducer Kulite EWCT-312 cooled (exhaust)
  - Piezo – electrical transducer Kistler 6052C (cylinder)
- **Crank angle position**
  - Optical encoder AVL 364, resolution:  $0.5^\circ$
- **Fuel mass – flow**
  - Mass flow meter Micromotion ELITE CMF025
- **Equivalence ratio**
  - Linear Lambda sensor NTK 6312-W1
- **Data acquisition system**
  - AVL Indicom 1.6

# Experimental Apparatus

- Crank angle based cylinder and duct pressure traces
- Measure of fuel mass flow and equivalence ratio:
  - Calculation of Volumetric Efficiency



Intake pressure (downstream throttle):  
*Cylinder 1*  
*Cylinder 4*  
Exhaust pressure:  
*Cylinder 4*

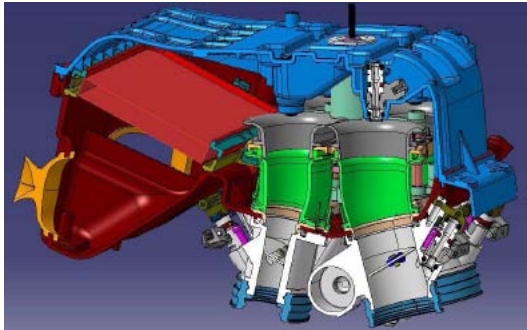


Equivalence ratio:  
*Cylinder 1*  
*Cylinder 2*  
*Cylinder 3*  
*Cylinder 4*



# Intake system analysis

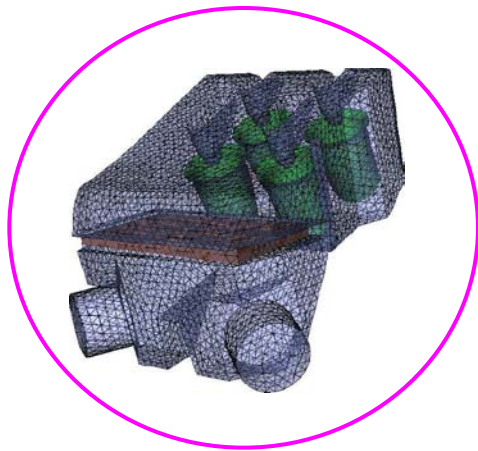
- Creation of 1D schematic and 3D mesh
- 1D and 3D model interfaces on boundary patches



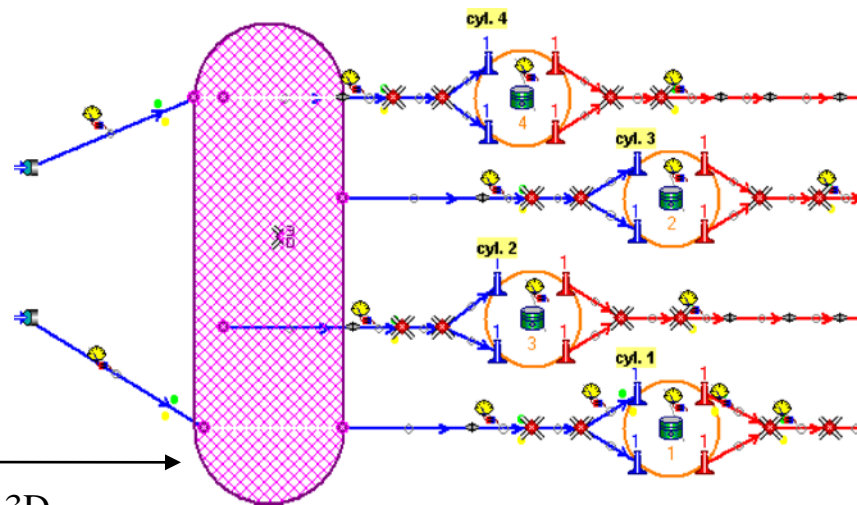
Engine intake system:

airbox, trumpets, throttle body and fuel injectors

Calculation domain  
from 3D CAD model



1D - 3D  
coupled model



Layout of 3D domain coupled with 1D  
model



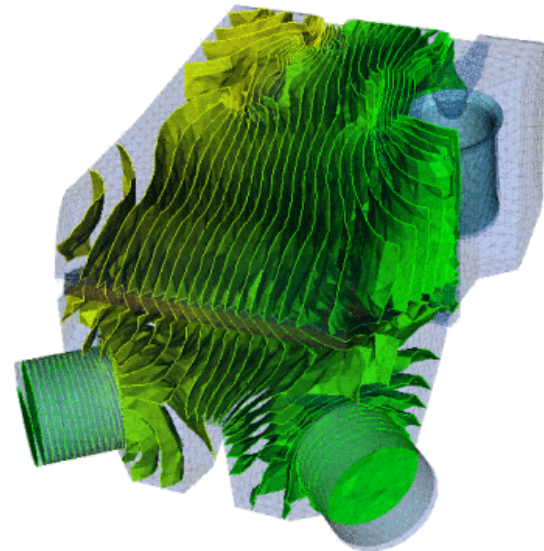
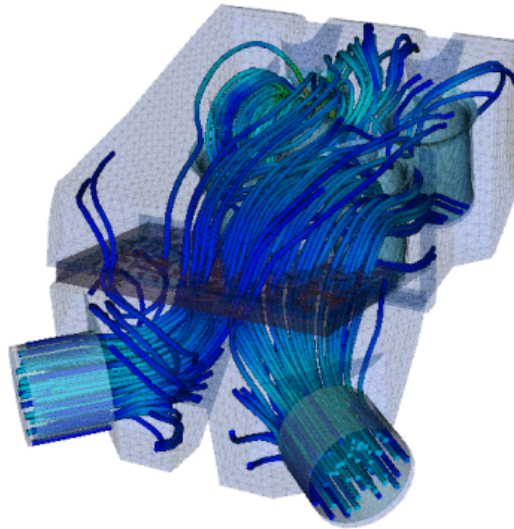
# Intake system analysis

- Multi-D analysis



Airbox mesh: full tetrahedral grid of about 57500 cells (mean side dimension: 5 mm).

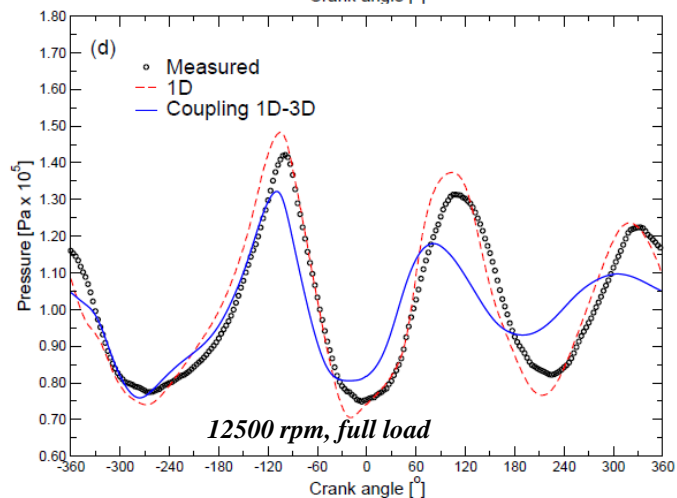
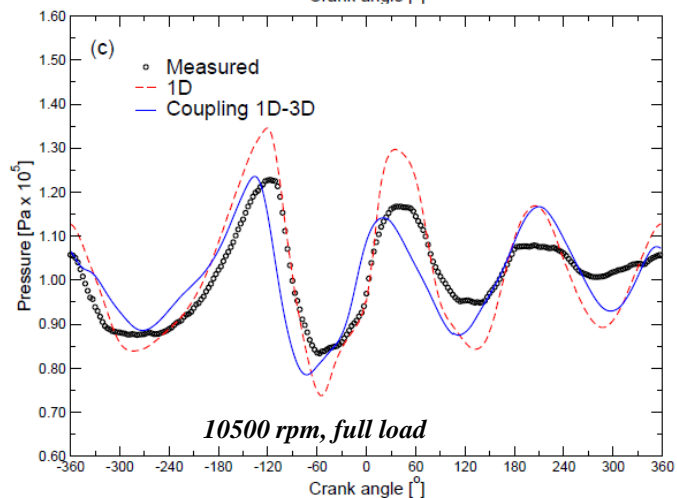
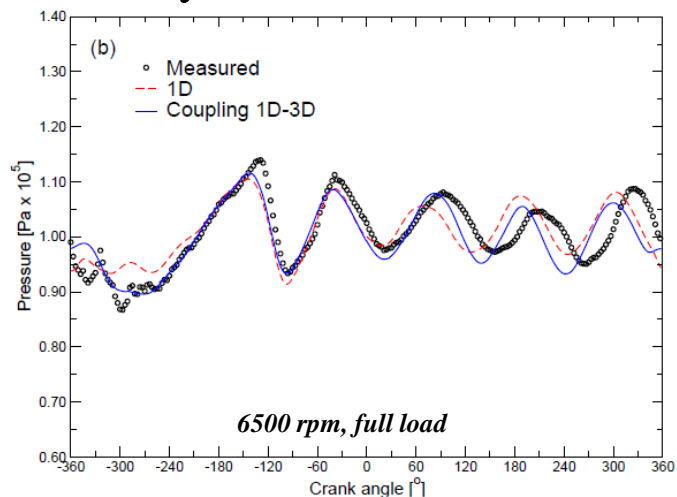
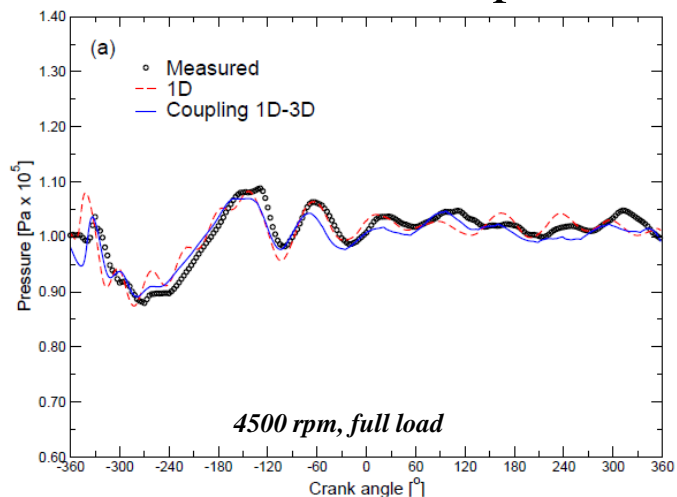
Air filter panel: airbox grid area of about 1700 cells.



Velocity and pressure fields at 12500 rpm

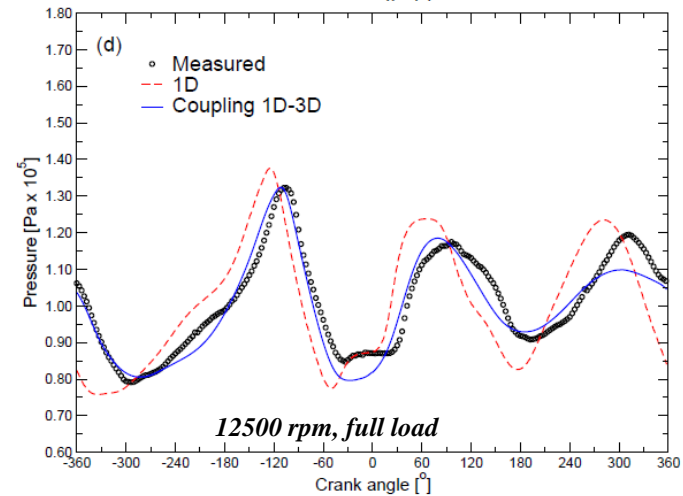
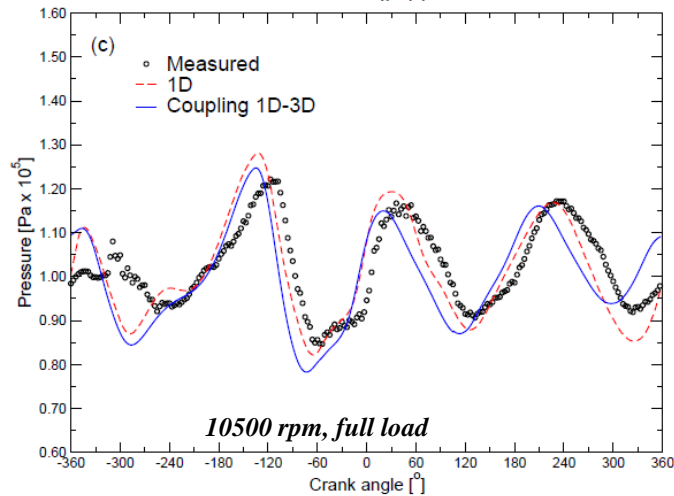
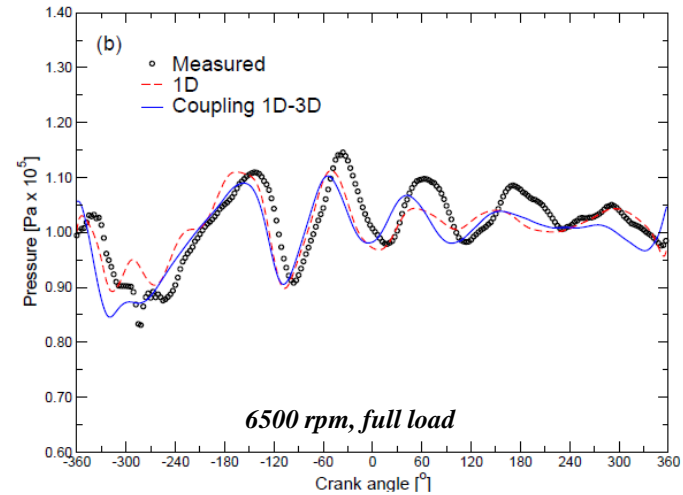
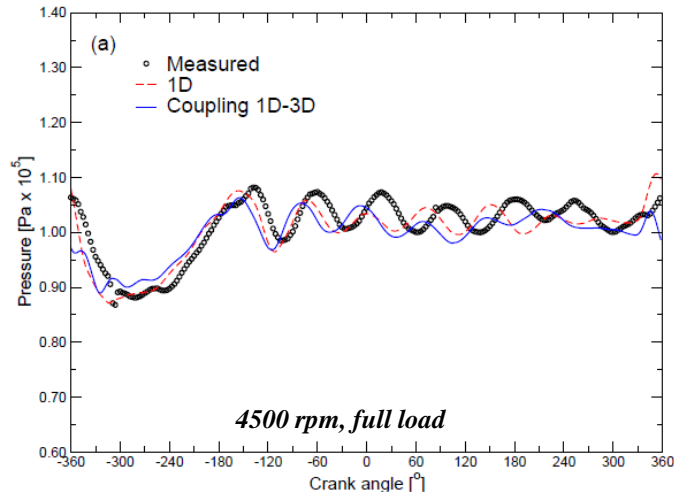
# Intake system analysis

- Simulations results: duct pressure upstream cylinder 1



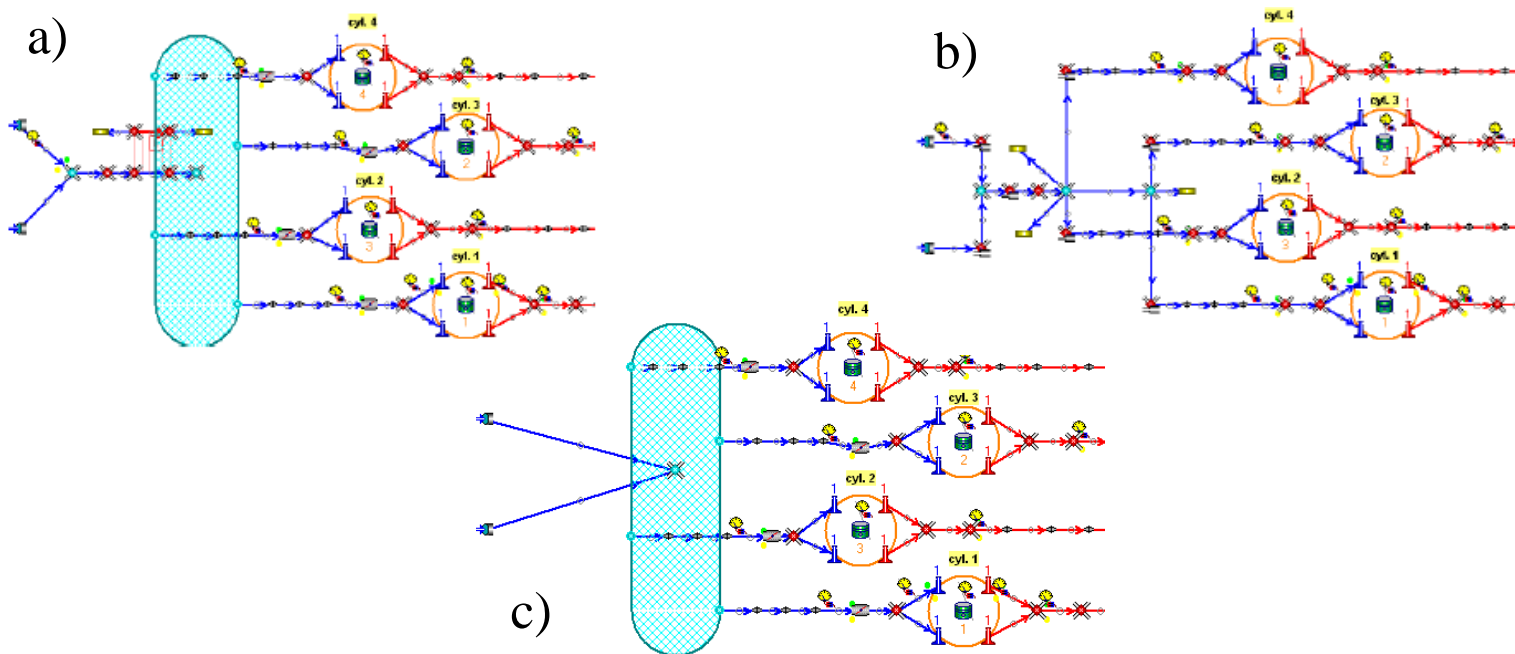
# Intake system analysis

- Simulations results: duct pressure upstream cylinder 4

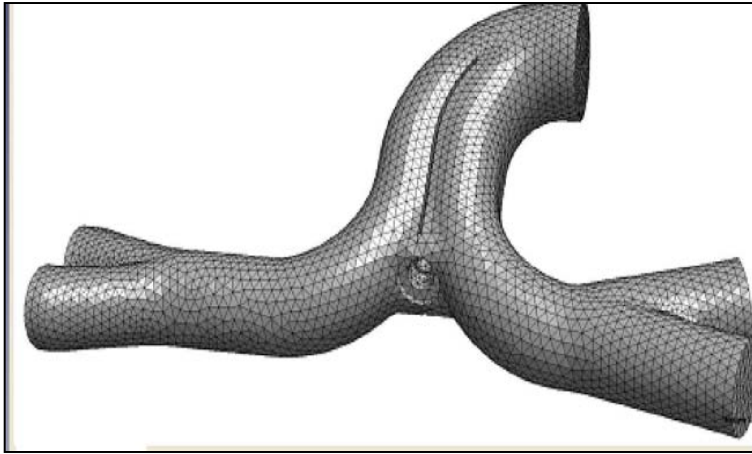


# Intake system analysis

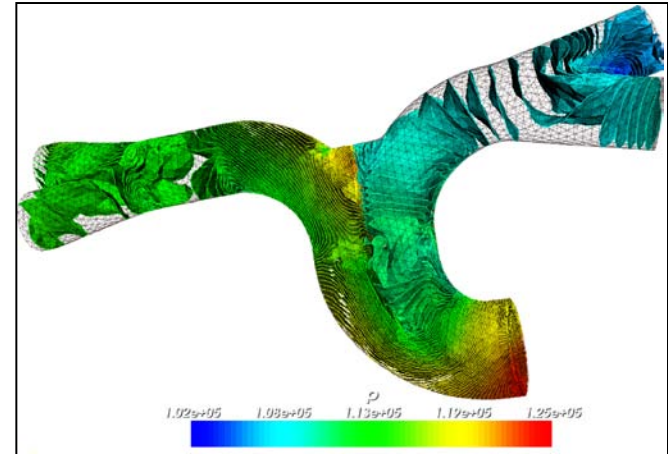
- 1D analysis
- Different modeling strategy
  - Prediction of fundamental resonances on principal axis:
    - a) transverse resonances; b) longitudinal resonances; c) 0D volume with corrective lengths on intake ducts



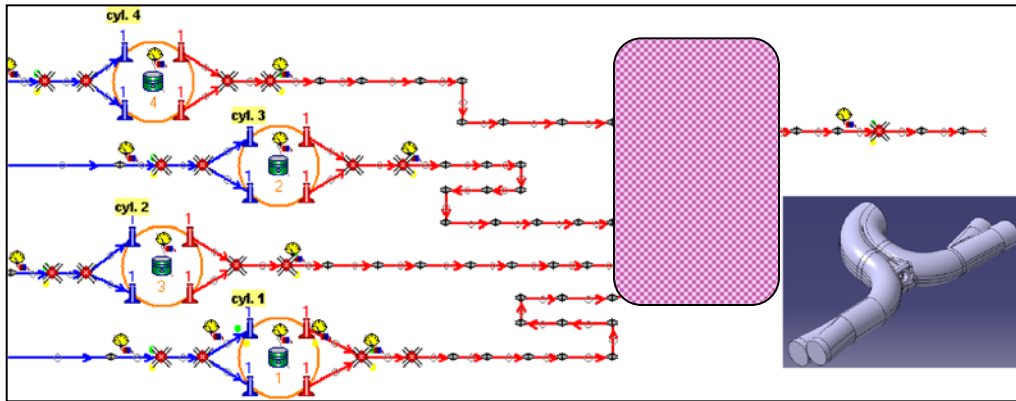
# Exhaust system analysis



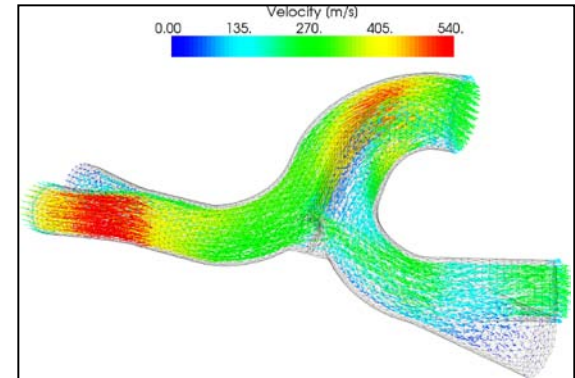
Exhaust junction mesh: full tetrahedral grid of about 36000 cells (mean side dimension: 5 mm)



Velocity and pressure fields at 12500 rpm

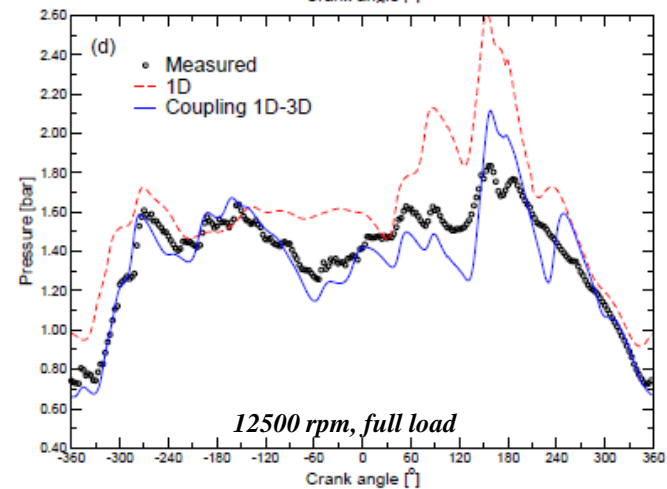
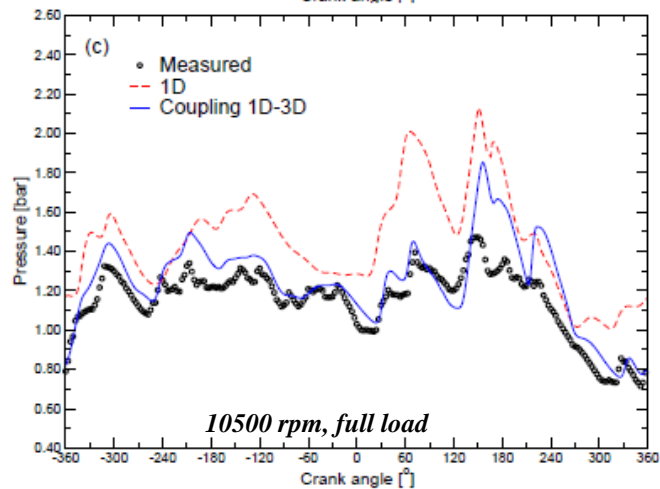
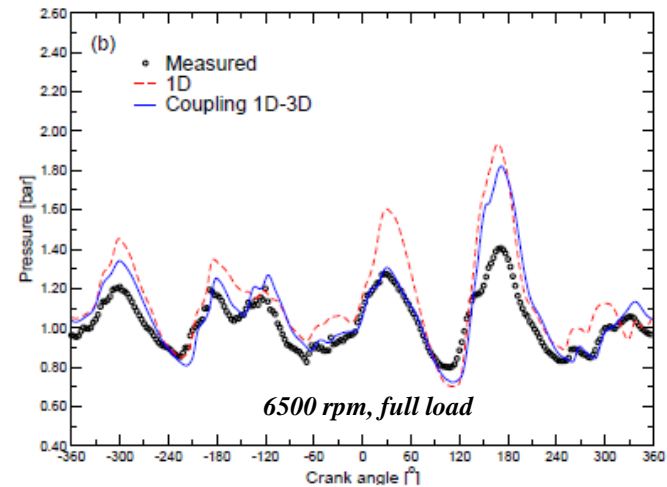
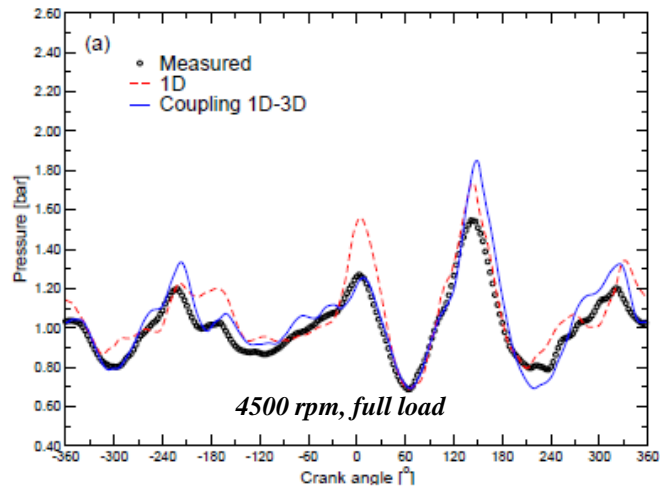


Exhaust 3D domain coupled with 1D model



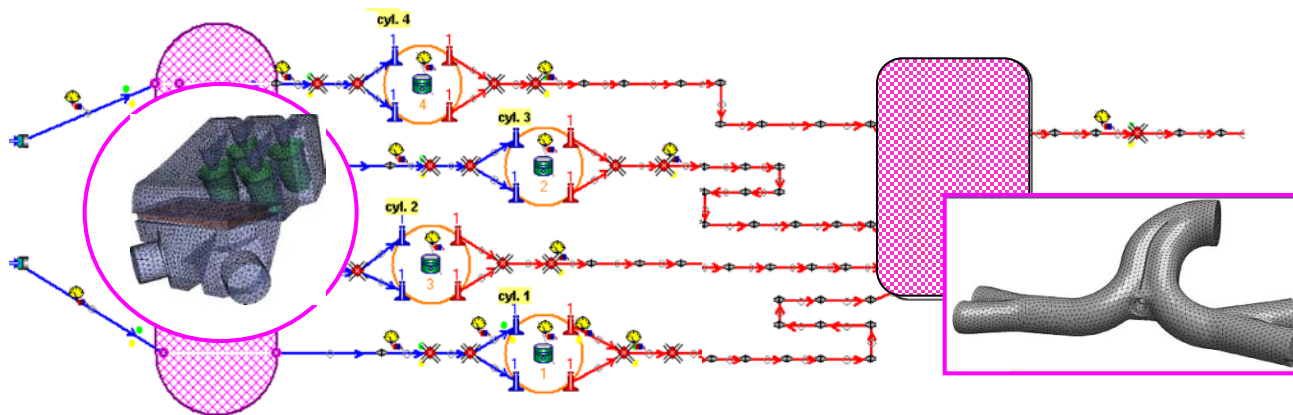
# Exhaust system analysis

- Simulation results: duct pressure downstream cylinder 4



# Prediction of volumetric efficiency

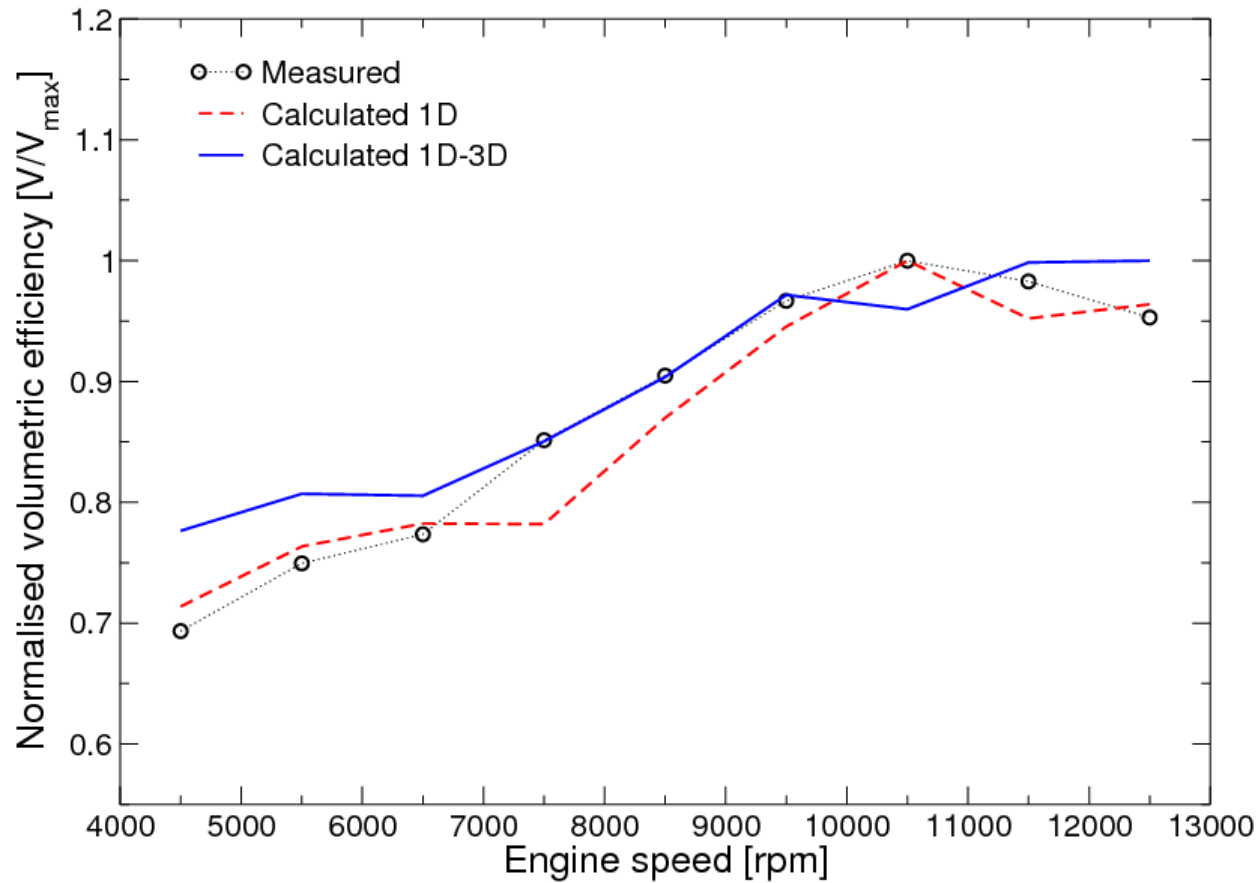
- 1D–3D simulation with both airbox and exhaust junction
- Coupling of multi-D domains
  - Engine revolution speed range:
    - (4500 ÷ 12500) rpm, step 1000 rpm
    - 5 ÷ 6 engine cycles for each engine revolution speed



- Calculation time: ~100 hours (~50 engine cycles) with AMD Athlon™ 64 X2 6000+ / 3 GHz CPU.

# Distribution of cylinder volumetric efficiency

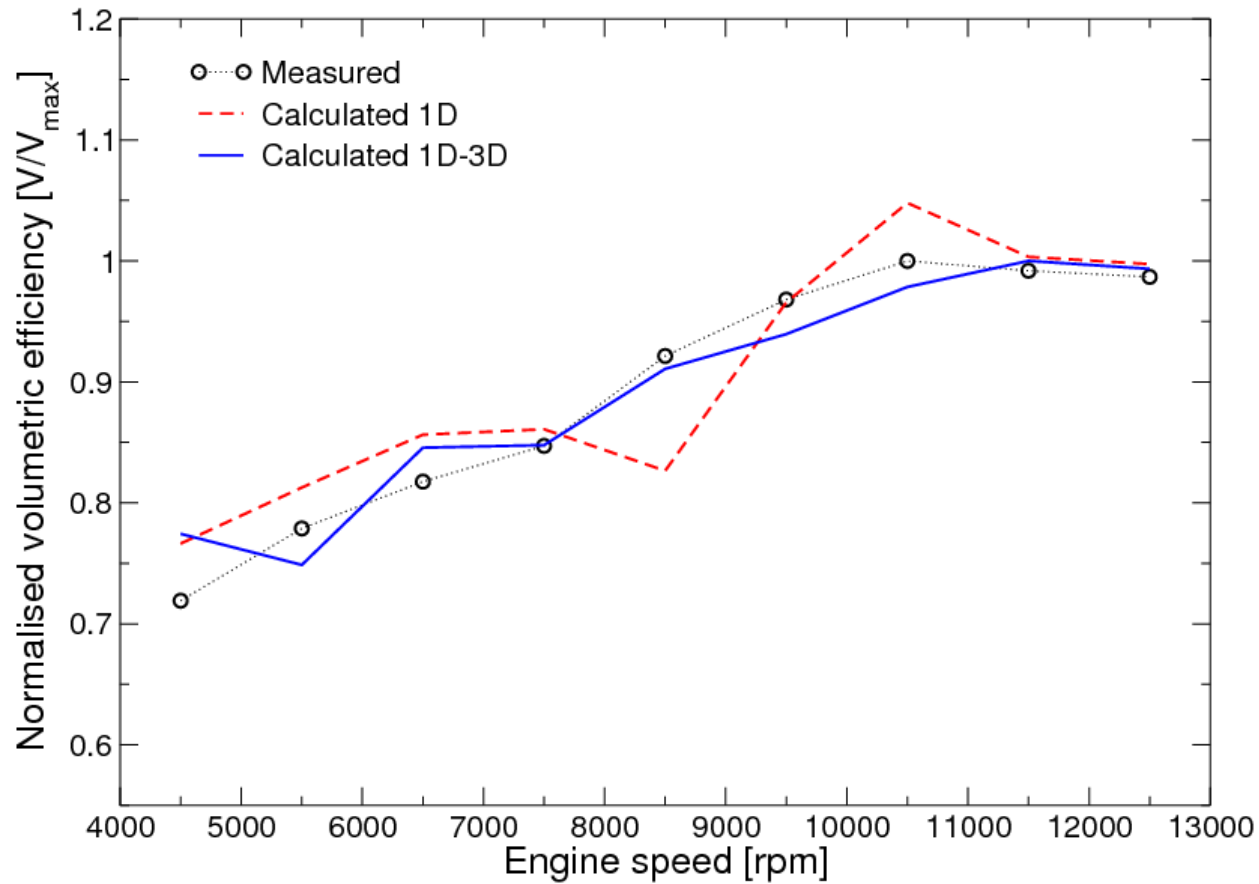
- Simulation results (full load): cylinder n.1





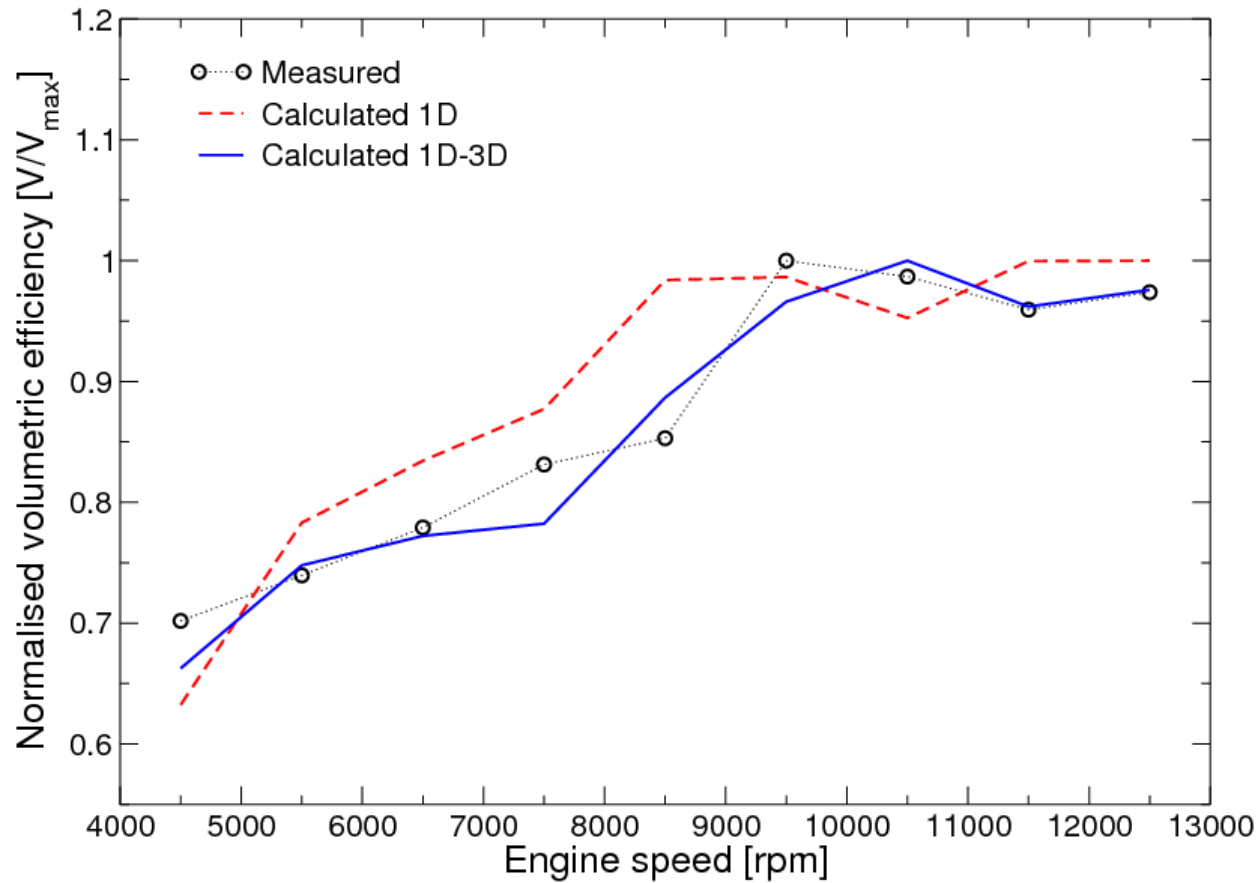
# Distribution of cylinder volumetric efficiency

- Simulation results (full load): cylinder n.2



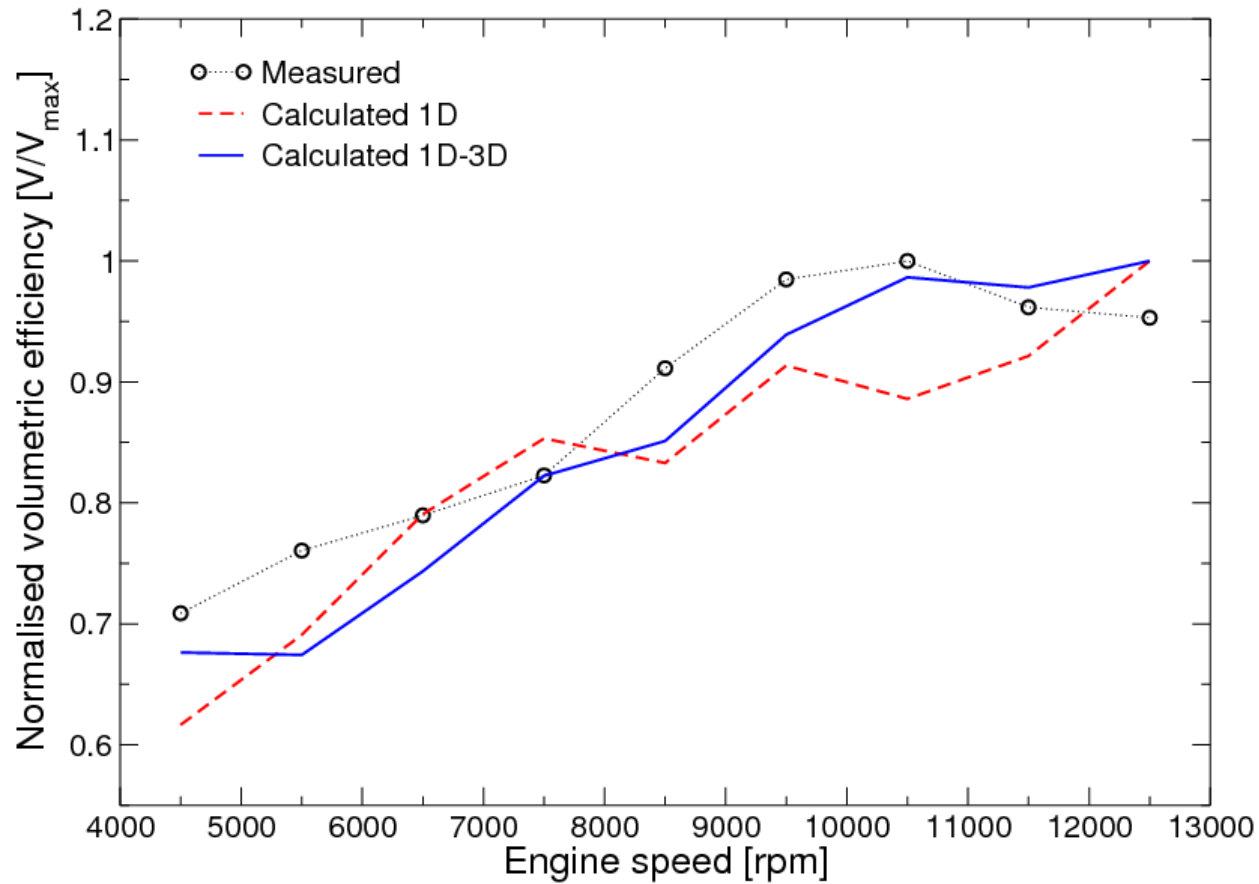
# Distribution of cylinder volumetric efficiency

- Simulation results (full load): cylinder n.3



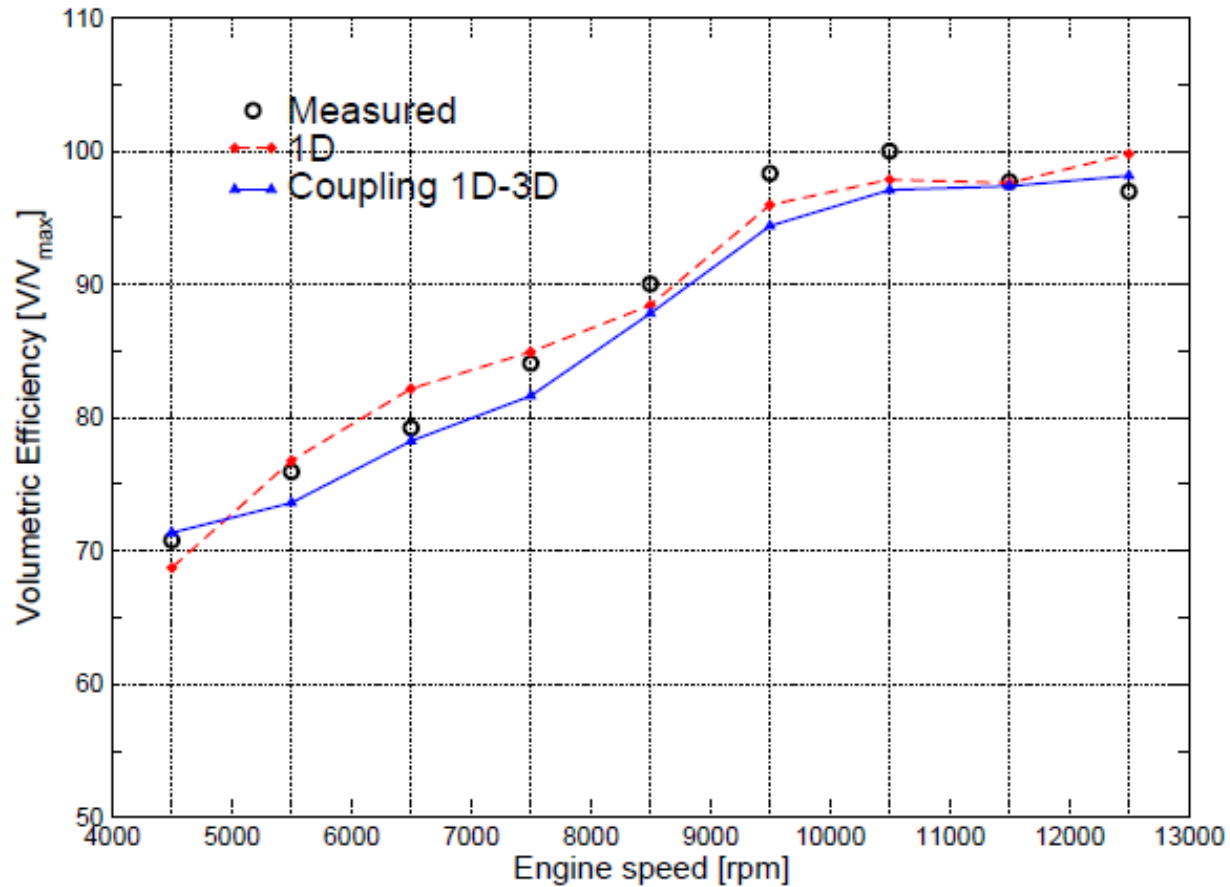
# Distribution of cylinder volumetric efficiency

- Simulation results (full load): cylinder n.4



# Prediction of volumetric efficiency

- Simulation results (full load): overall values

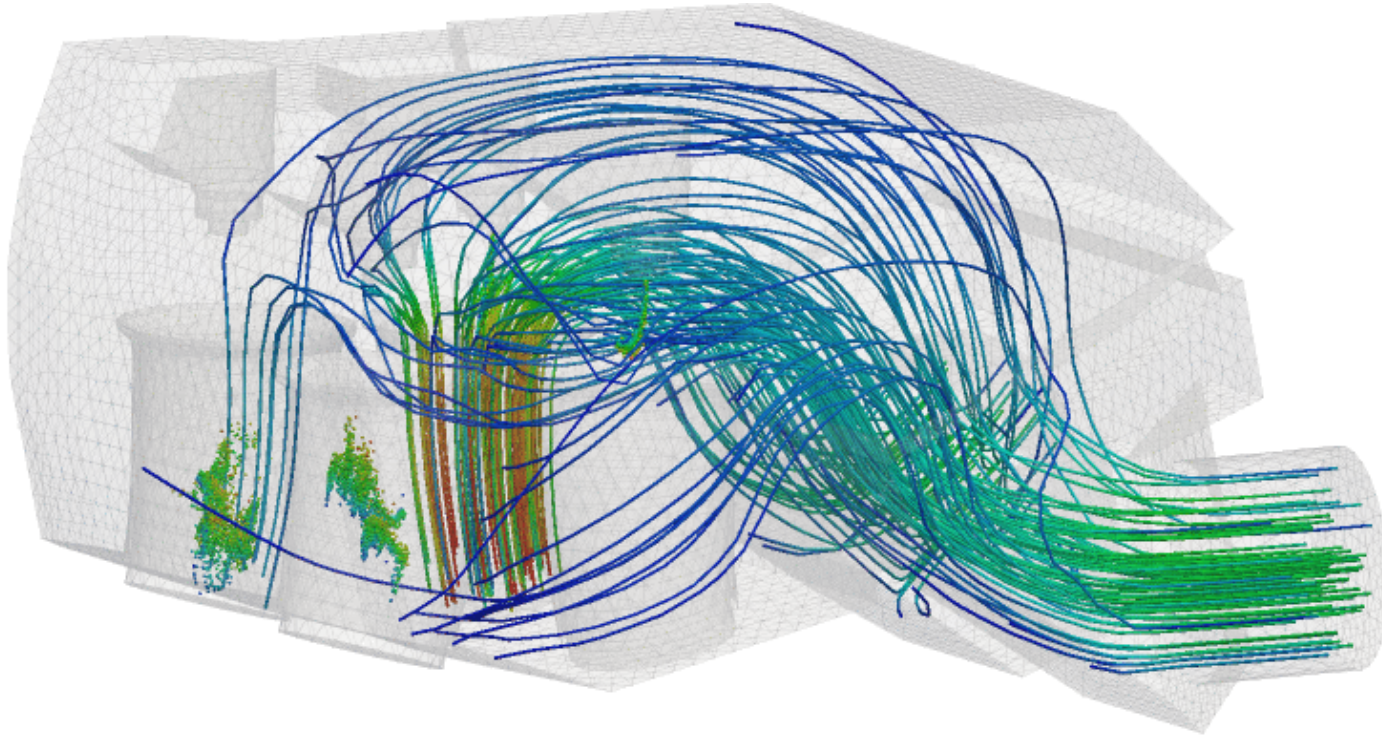


# Conclusions

- Detailed fluidynamic analysis to better understand the phenomena in complex shape intake and exhaust systems of a high performance engine.
- Different approaches to reach this target: 1D analysis and 1D–multiD integrated analysis.
- Good results achieved with each approach in the prediction of fluid dynamic parameters of the engine
  - 1D: low CPU time and good prediction, but corrective lengths are required
  - 1D-3D: high CPU time and very good prediction, independent from the geometrical complexity of the calculation domain.

# Work in progress and future developments

- A new coupled solver to simulate spray evolution into the intake system, in order to predict the effect of the flow field on the mixture formation.



- A new solver is under development to take model the liquid film formation on the airbox walls



**Thank you for your attention**

